



US007246614B2

(12) **United States Patent**
Huggins

(10) **Patent No.:** **US 7,246,614 B2**
(45) **Date of Patent:** **Jul. 24, 2007**

(54) **LIGHTWEIGHT, PORTABLE COOKING STOVE**

(76) Inventor: **Timothy Gregg Huggins**, 5215A Fleetwood Oaks, Dallas, TX (US) 75235

(*) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 219 days.

1,194,131 A *	8/1916	Brown	126/29
1,210,099 A	12/1916	Pohl	
1,373,828 A *	4/1921	Nottingham	126/38
1,421,209 A *	6/1922	Gehrer	431/249
1,437,073 A	11/1922	Popper	
1,437,812 A *	12/1922	MacMillan	126/38
1,487,474 A *	3/1924	Rhodes	126/38
1,508,334 A *	9/1924	Ingalls	126/43

(Continued)

FOREIGN PATENT DOCUMENTS

(21) Appl. No.: **10/951,128**

(22) Filed: **Sep. 27, 2004**

GB	0 232182	*	6/1925
GB	0 527286	*	10/1940
GB	2 228 077	*	8/1990

(65) **Prior Publication Data**

US 2006/0065264 A1 Mar. 30, 2006

(51) **Int. Cl.**

F23D 5/04	(2006.01)
F23D 5/12	(2006.01)
F23D 5/18	(2006.01)
F24C 5/02	(2006.01)

(52) **U.S. Cl.** **126/44**; 126/43; 126/9 R; 126/9 B; 126/39 R; 126/40; 126/50

(58) **Field of Classification Search** 126/9 R, 126/9 B, 25 R, 38, 39 R, 41 R, 40, 50, 43, 126/44

See application file for complete search history.

(56) **References Cited**

U.S. PATENT DOCUMENTS

26,595 A *	12/1859	Danner et al.	126/38
79,084 A *	6/1868	Smallwood	126/38
500,765 A *	7/1893	Heller	126/43
546,632 A *	9/1895	Wojdkow	431/323
560,319 A	5/1896	Mast	
766,618 A	8/1904	Heinrichs	
893,559 A *	7/1908	Yates	431/230
936,482 A *	10/1909	Sheeley	126/43
1,135,317 A *	4/1915	Patterson	431/205

OTHER PUBLICATIONS

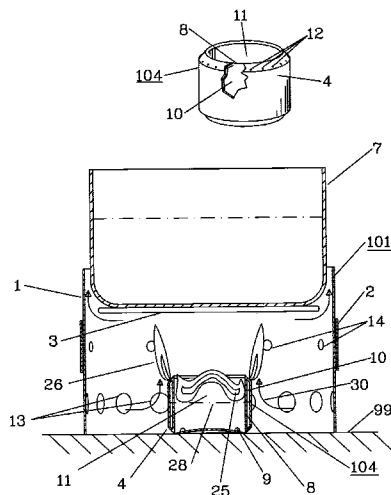
[http:// www.trangia.se](http://www.trangia.se), *trangia_stoves_series 27-1*, Trangia Corporate Web Site, Nov. 17, 2004.
http://en.wikipedia.org/wiki/soda_can_stove, *Soda Can Stove*, Wikipedia Web Site, Dec. 20, 2004.

Primary Examiner—Carl D. Price
(74) *Attorney, Agent, or Firm*—Michael A. O’Neil

(57) **ABSTRACT**

A small, lightweight, portable cooking stove that utilizes alcohol as fuel and is suitable for backpacking, hunting, camping and similar activities. The cooking stove includes a combustion chamber which encloses, contains and regulates the entire combustion process. The stove also includes an efficient fuel vaporizer, which is adapted to perform in unison with the combustion chamber, to effect the metered and efficient mixing of the air and fuel vapor. An integral, simple and convenient means is provided to vary the intensity and pattern of the heat output, thereby controlling the cooking performance of the stove. The stove achieves high heat output and efficiency, low fuel consumption, and superior cooking performance in a small, lightweight, portable, convenient, simple and integrated assembly.

2 Claims, 8 Drawing Sheets



U.S. PATENT DOCUMENTS

2,088,797 A	8/1937	Larsen		4,539,973 A	9/1985	Hait	
2,154,305 A *	4/1939	Goerl	126/38	4,604,053 A	8/1986	De La Rosa	
2,174,824 A *	10/1939	Frank	126/266	4,722,322 A *	2/1988	Varney et al.	126/261
2,681,056 A *	6/1954	Fischl	126/43	4,793,321 A *	12/1988	Rafford et al.	126/43
2,877,759 A	3/1959	Giese		4,905,659 A	3/1990	Armistead	
3,057,399 A *	10/1962	Vant	431/37	4,909,235 A	3/1990	Boetcker	
3,315,723 A *	4/1967	Snodgrass et al.	431/206	4,915,091 A *	4/1990	Varney	126/65
3,359,963 A *	12/1967	Kostial	126/9 R	5,163,415 A	11/1992	Moncrief	
3,371,659 A	3/1968	Paspalas		5,284,126 A *	2/1994	Varney	126/50
3,381,678 A	5/1968	Fry		5,842,463 A *	12/1998	Hall	126/9 R
3,566,857 A *	3/1971	Price	126/262	5,881,709 A	3/1999	Daoust	
3,574,505 A	4/1971	Kimball		5,915,371 A *	6/1999	Hering	126/9 R
3,726,268 A *	4/1973	Stansell	126/262	5,979,428 A *	11/1999	Greene, Jr.	126/38
4,164,930 A	8/1979	Johnston		6,213,115 B1 *	4/2001	Anderson et al.	126/46
4,508,094 A	4/1985	Hait		6,488,023 B2	12/2002	Pecoskie	

* cited by examiner

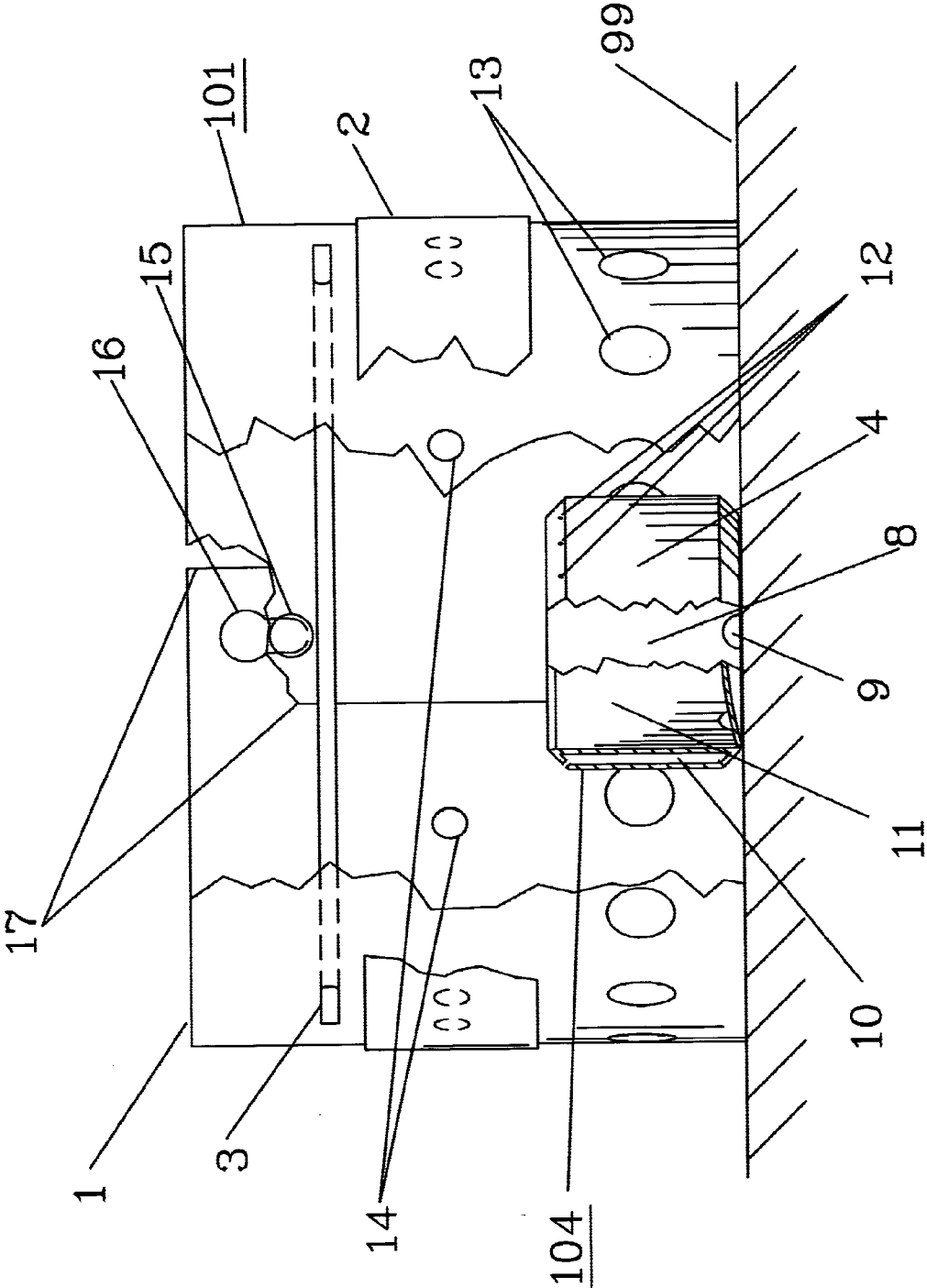


FIG. 1

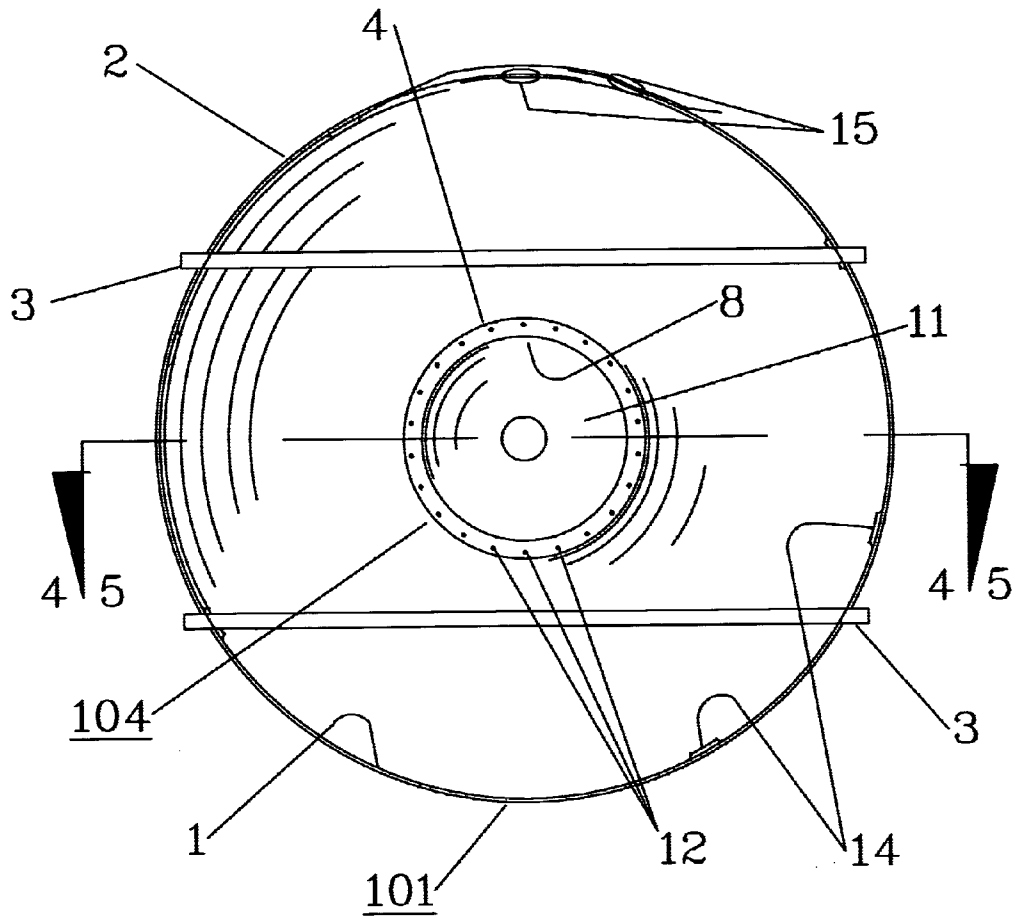


FIG. 2

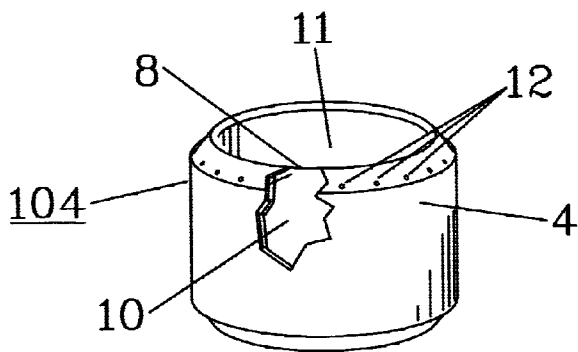


FIG. 3

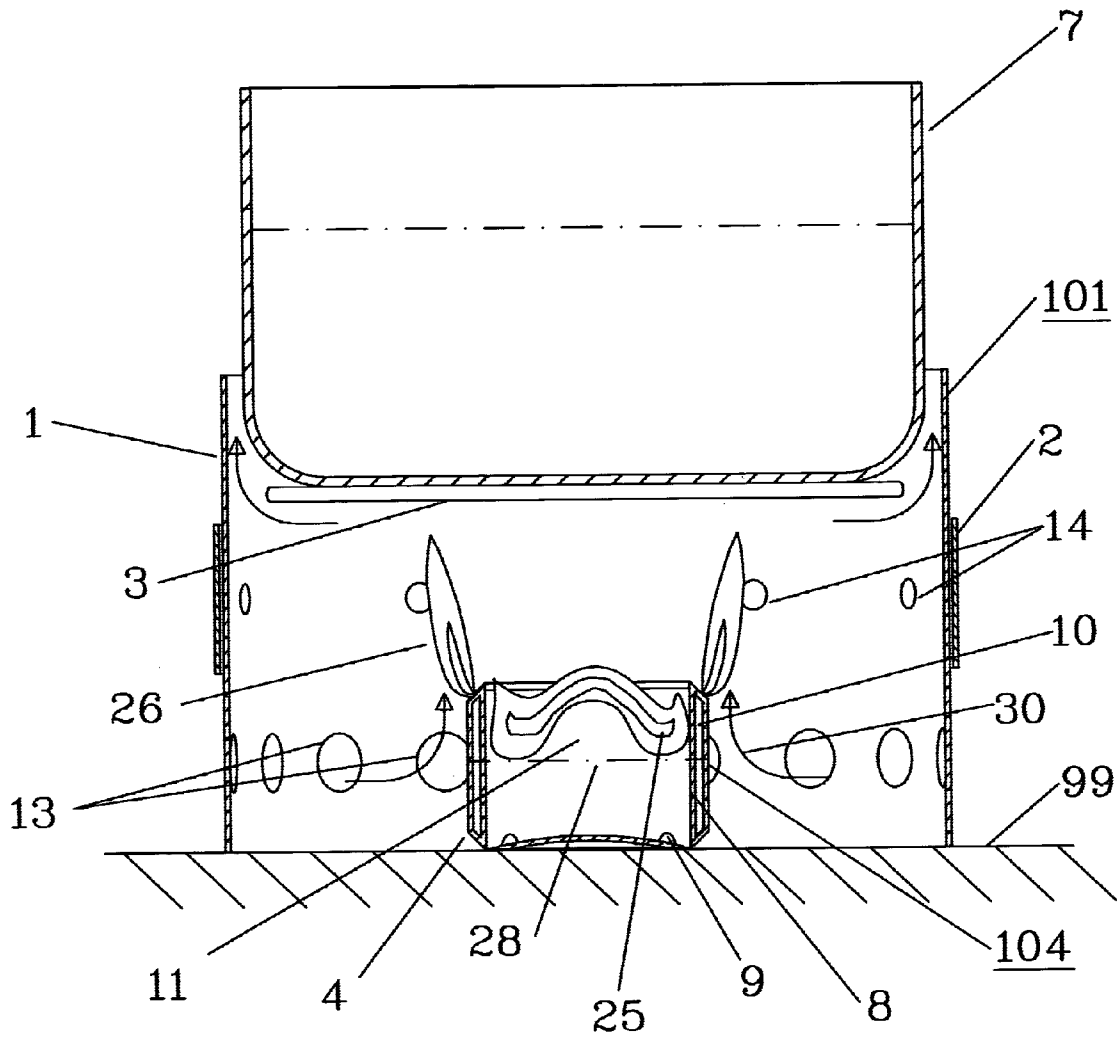


FIG. 4

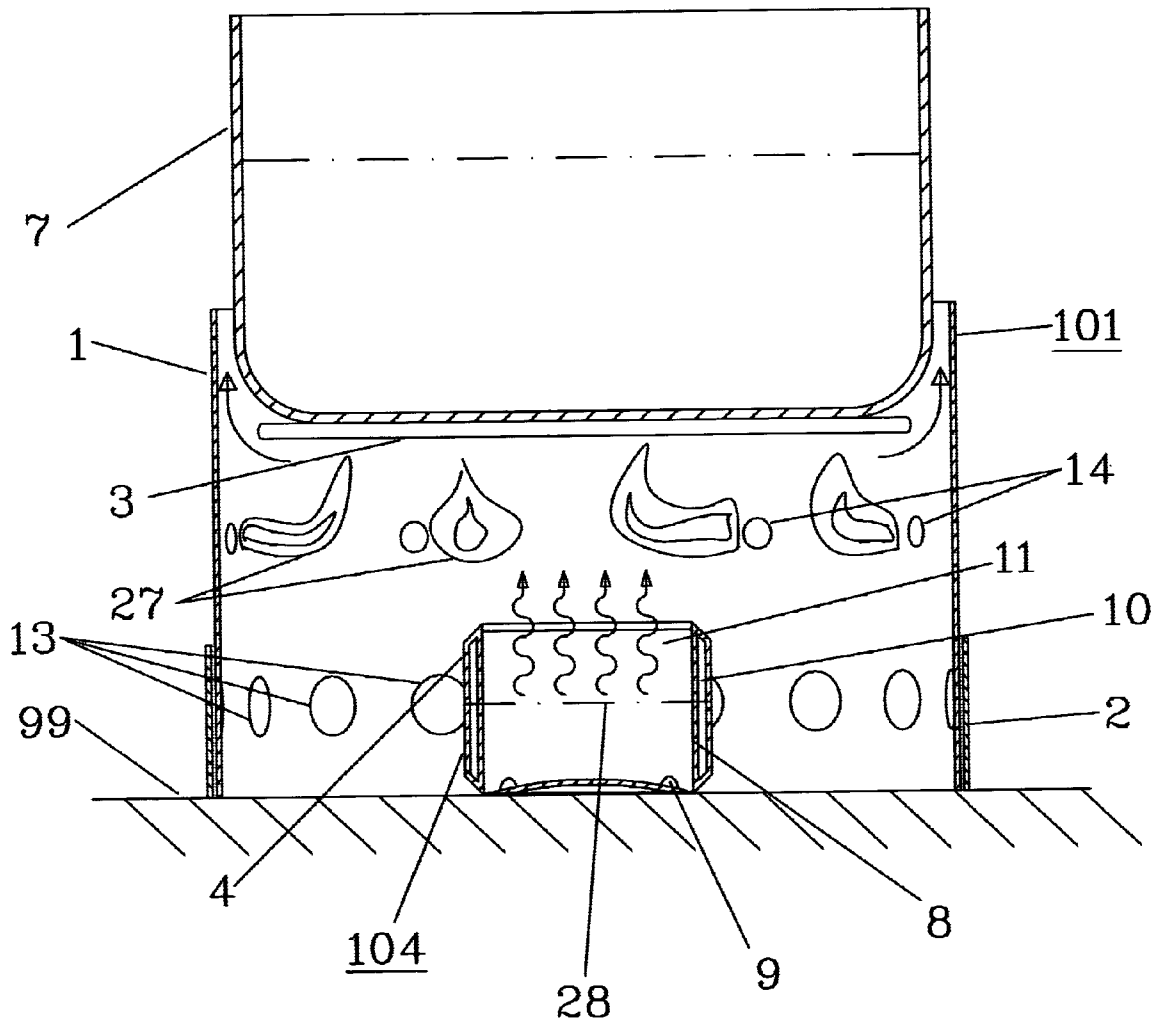


FIG. 5

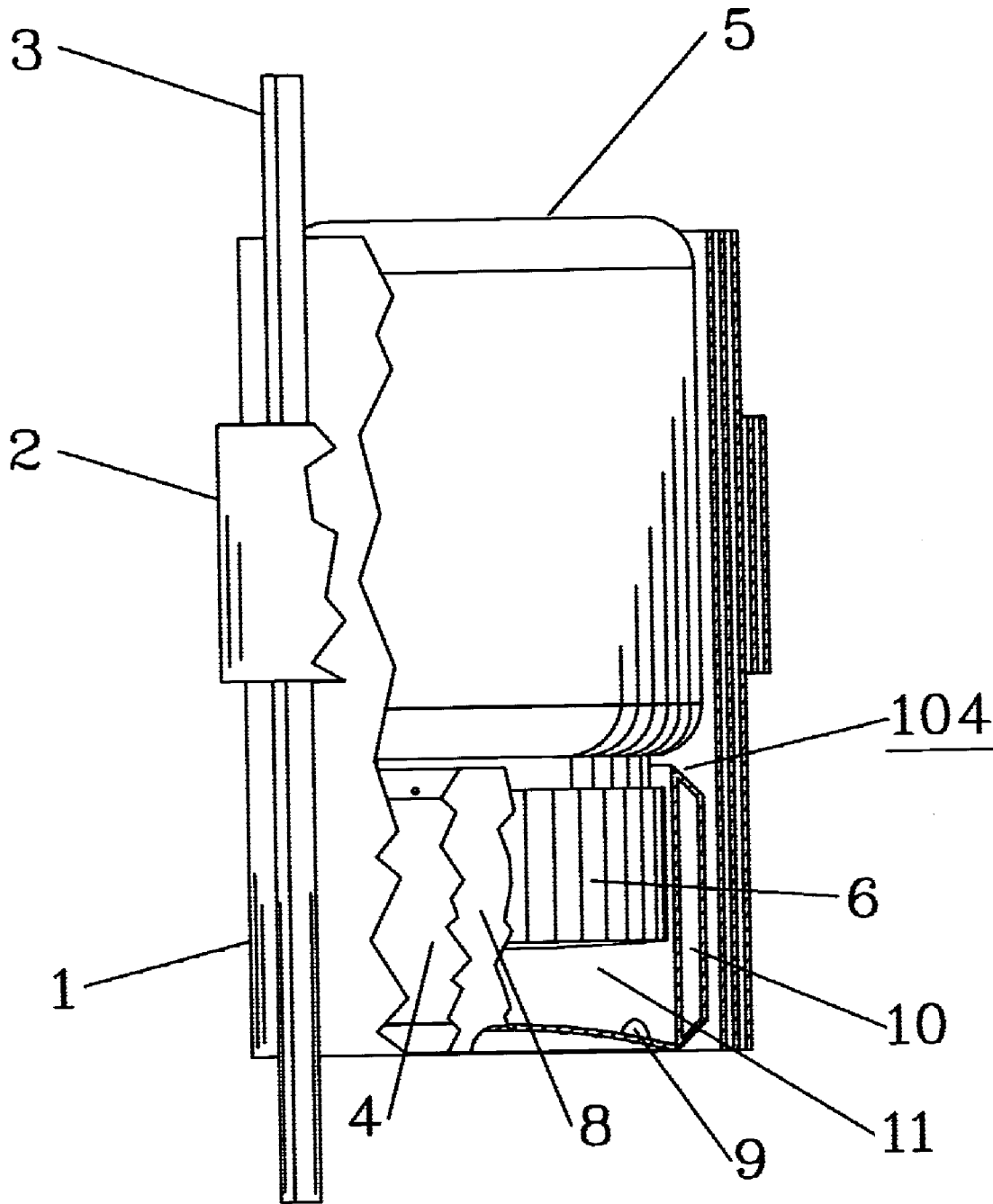


FIG. 6

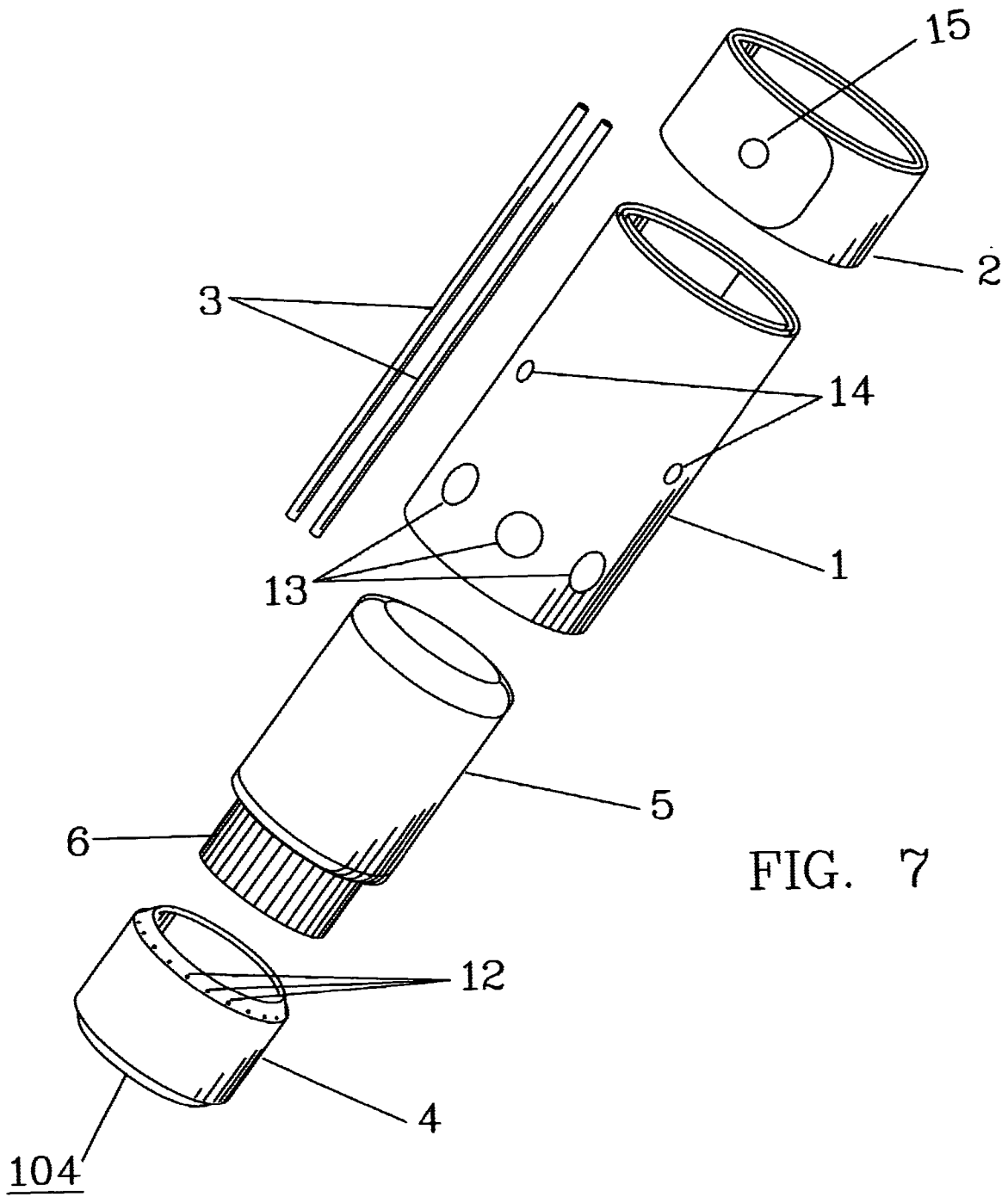


FIG. 7

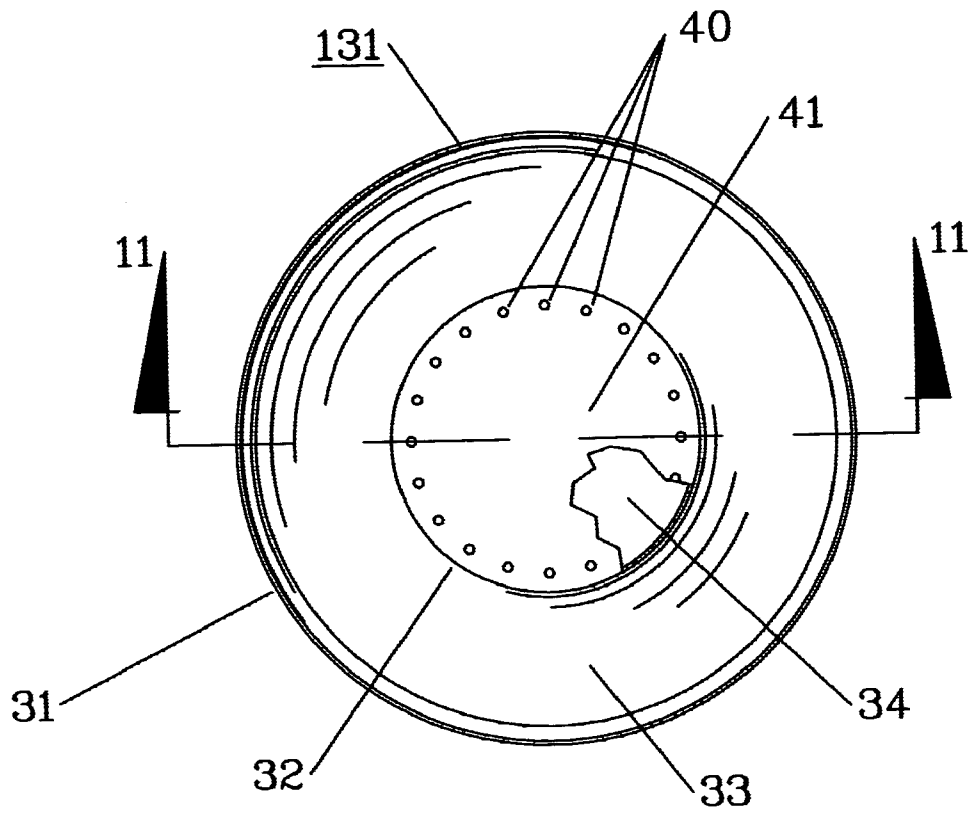


FIG. 8

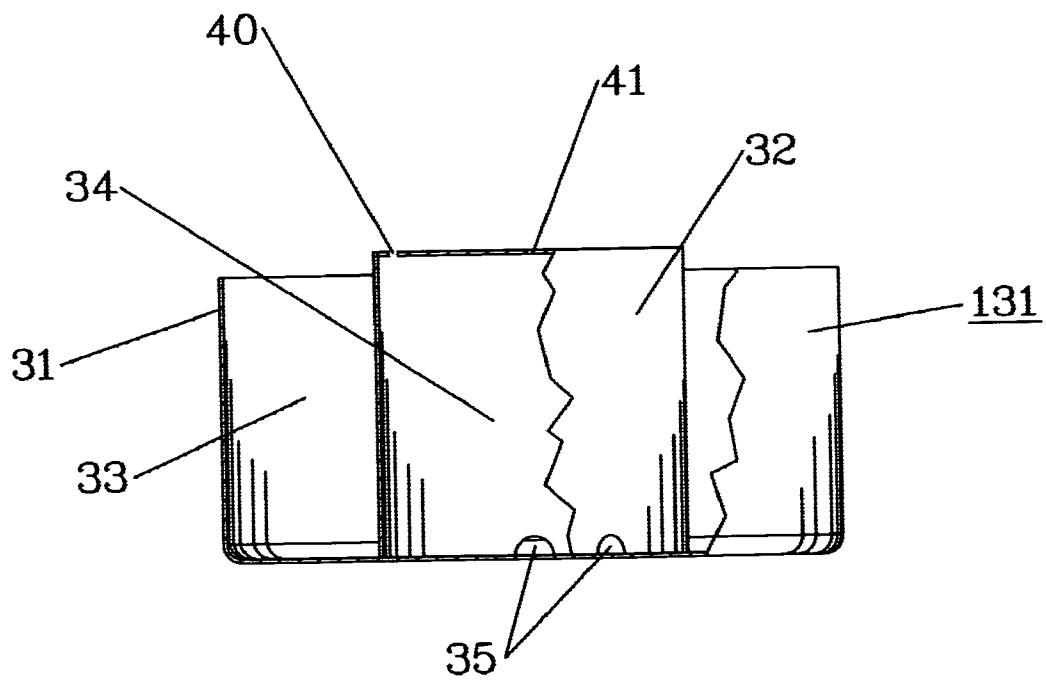


FIG. 9

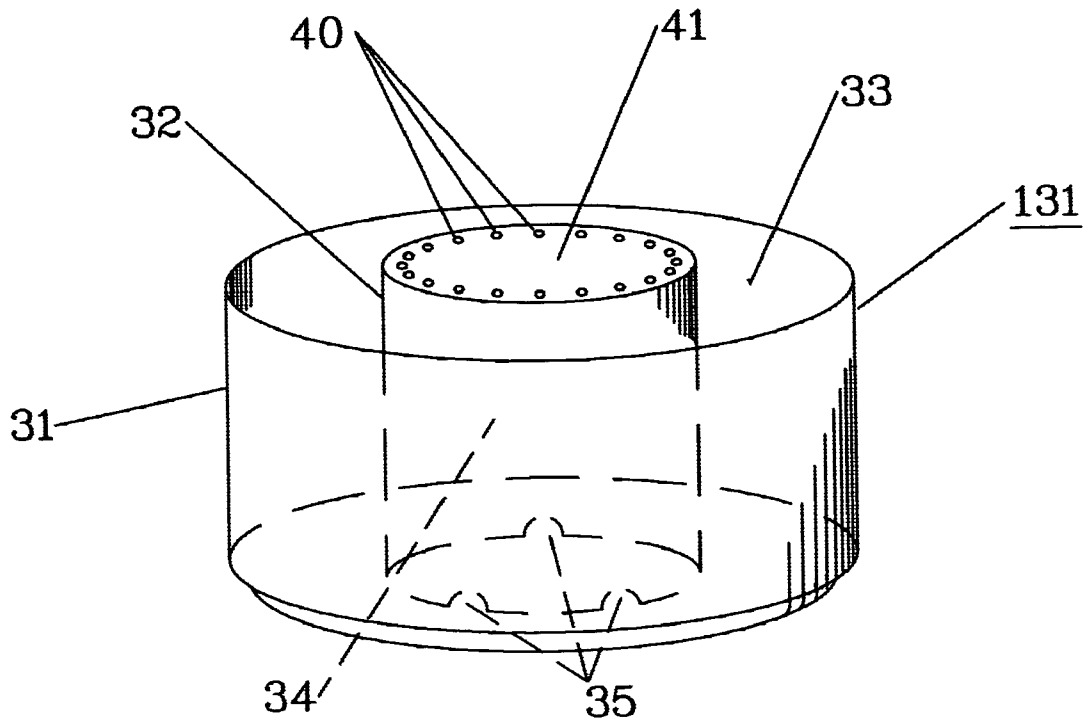


FIG. 10

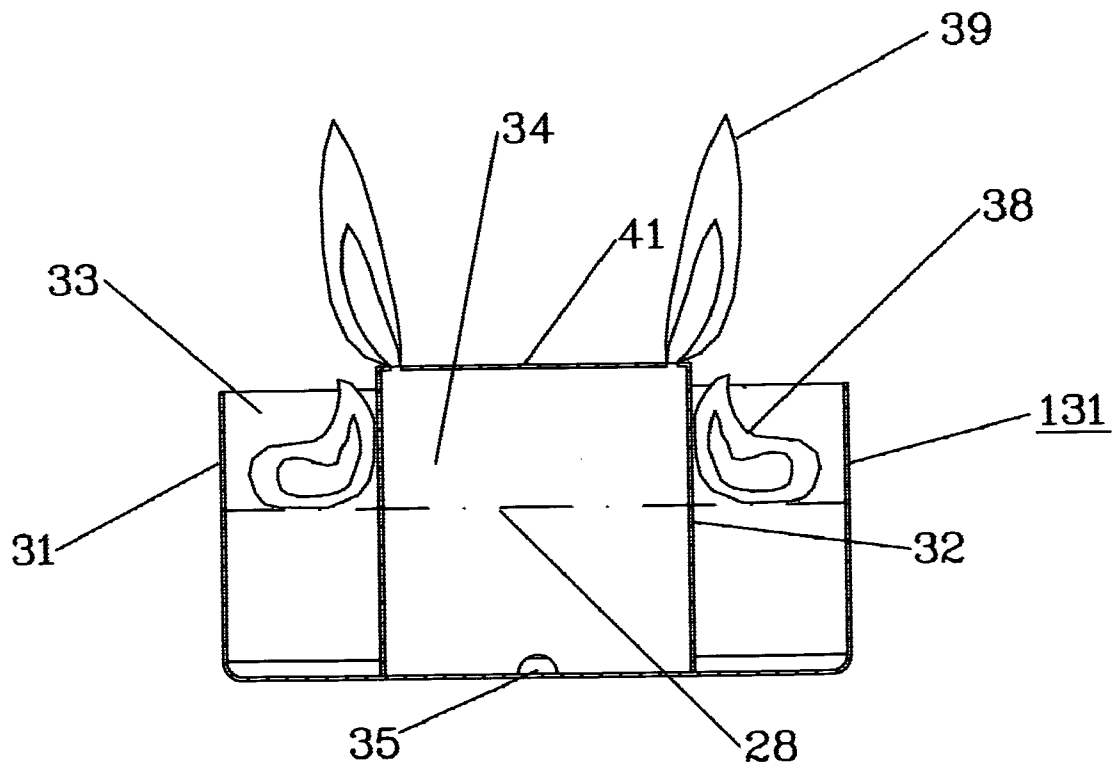


FIG. 11

1

**LIGHTWEIGHT, PORTABLE COOKING
STOVE**2 CROSS REFERENCE TO A RELATED
APPLICATION

Not Applicable

3 STATEMENT REGARDING FEDERALLY
SPONSORED R&D

Not Applicable

4 REFERENCE

Not Applicable

5 BACKGROUND

1. 5.1 Field of the Current Invention

The invention is a small, lightweight, portable cooking stove that uses alcohol as fuel.

2. 5.2 Prior Art

Portable cooking stoves designed to use alcohol as a fuel are well known to prior art. These are used by hikers, campers, backpackers, hunters and others who have a need to boil water or cook a hot meal in remote locations or primitive conditions. As a fuel, alcohol has various advantages over petroleum based fuels. Alcohol is widely available, inexpensive, convenient, relatively innocuous and easy to handle. It can be readily repackaged and safely carried in lightweight plastic bottles. Alcohol stoves are typically smaller and lighter than petroleum-fueled stoves. Alcohol stoves are typically also very simple, reliable and easy to use, and have few, if any, moving parts. Because of these advantages, alcohol-fueled stoves are very popular in many parts of the world. However, in the United States they have had limited popularity. This limited popularity is primarily due to two factors.

The first factor that has limited the popularity of alcohol-fueled stoves in the United States is their overall poor performance. Under the best conditions, the heat output from these stoves is marginal. In real outdoor conditions of wind and weather, these stoves rapidly become incapable of cooking a meal. These stoves demonstrate a variety of forms and features that cause inefficient performance and wasted heat. The overall inefficient performance of alcohol stoves known to prior art also causes an excessive consumption of fuel. This is undesirable both due to the increased cost of operating the stove, as well as the increased weight of fuel that must be carried.

The second factor that has limited the popularity of alcohol-fueled stoves in the United States is the inability to effectively adjust the heat output of the stove. Often these stoves provide no manner of heat adjustment. When they do, it is commonly effected by means of some type of snuffer in the form of a partial lid, cap or cover that is positioned in such a way that partially interferes with or throttles the flame. This method of adjusting the heat output has several disadvantages. Positioning or adjusting of the snuffer can be a rather awkward undertaking and requires removing the cooking pot or reaching under it. This presents a danger of burning or scalding. Often the cooking pot must be completely removed from the stove and set aside while positioning the snuffer. Obstructing the flame in this way tends to produce a small hot spot on the bottom of the cooking pan and prevents the even distribution of heat for thorough and rapid cooking.

In summation, the two primary factors that have limited the popularity of alcohol stoves in the United States are their

2

overall poor performance and the inconvenience with which the heat adjustment is effected. Similarly, petroleum-fueled stoves enjoy a broad popularity primarily due to their advantages of high heat output and the ability to adjust this heat output for control of the cooking process. These advantages are generally viewed as outweighing their numerous disadvantages. The disadvantages include the need for bulky and heavy metal fuel containers, high operating cost, poor operation in wind or cold weather, need to constantly tend them during cooking, odor, danger of fuel spills, complexity, poor reliability, the need to carry maintenance kits, and the possibility of dangerous flare-ups when lighting the stove. In light of these numerous disadvantages, the importance of high heat output and adjustable cooking performance in determining the overall utility and popularity of a stove is clearly seen.

6 SUMMARY OF THE CURRENT INVENTION

6.1 Objectives

The current invention seeks to achieve a variety of improvements over the portable cooking stoves which are known to prior art. The current invention is conceived and designed with the intent of achieving specific objectives for enhanced performance and convenience over the prior art.

It is an objective of this invention to achieve a high heat output and stable cooking performance suitable for fast and effective cooking in a variety of outdoor conditions of wind and weather—attaining overall cooking performance which consistently meets or exceeds the performance of petroleum-fueled stoves.

It is a further objective of this invention to achieve a high fuel efficiency, in order to minimize operating costs and the weight of fuel that must be carried.

It is a further objective of this invention to provide a convenient and simple means for adjusting the heat output of the stove to effectively control the cooking process.

It is a further objective of this invention to be small, lightweight and conveniently portable. When packed with a fuel bottle, the entire stove shall be of a size that can be held in the palm of the hand and weigh only a few ounces. The entire invention is rendered conveniently portable in a sturdy and compact package.

It is a further objective of this invention to be largely trouble free, simple, reliable and easy to use.

It is a further objective of this invention that it will cool quickly so that it can be handled and stowed soon after use.

To achieve the above listed objectives, a portable cooking stove is described wherein the several components of the stove are adapted to perform together as an integrated unit—all components being engineered to operate in balanced synergy to maximize the overall performance and utility of the stove. The several components of the stove are adapted to work in concert to optimize as far as practicable the various fluidic, thermodynamic and heat transfer processes of the stove in a manner that is both unobvious and unknown to prior art.

6.2 Contents

6.2.1 Combustion Chamber

Stoves known to the prior art are typically intended to effect the combustion process in the open air. Because the combustion process takes place in the open air, there is no means for controlling the quantity of air in contact with the combustion process. This produces a condition known in the field of thermodynamics as “excess air”. The condition of excess air occurs when a combustion process is provided with more air than is required for the complete combustion

of the fuel. This excess air removes heat from the combustion process, thereby reducing the efficiency of the combustion process.

The importance and benefits of providing a combustion chamber to create a controlled volume wherein the entire combustion process can be enclosed, contained and encompassed in order to maximize the heat output of a cooking stove are well known and detailed in prior art. U.S. Pat. Nos. 5,915,371 and 5,842,463 both describe the functioning and the performance benefits derived from the use of a combustion chamber to maximize the heat output of a cooking stove. These inventions even describe a means for adjusting the draft to alter the airflow through the combustion chamber, thereby varying the heat output and adapting to various cooking requirements.

These inventions describe combustion chambers that are intended to be used primarily with solid fuel, such as wood or charcoal. They are adapted to address the specific problems associated with the combustion of such solid fuels. These inventions are not engineered nor adapted to meter and direct the flow of combustion air to efficiently mix with and effect the combustion of a vaporized fuel. These prior inventions do not conceive of the combustion chamber as integral with a specific combustion source, such that both components might be optimized and adapted to operate in balanced synergy. The cooking stove of the current invention incorporates a combustion chamber that is specifically engineered and adapted as a component within an integrated assembly, to meter and direct the flow of combustion air such that it efficiently mixes with and effects the combustion of a vaporized fuel.

The portable, alcohol-fueled cooking stove of the current invention comprises a combustion chamber which is engineered and adapted to create a controlled volume wherein the entire combustion process can be enclosed, contained and encompassed. The purpose of this combustion chamber is to meter, regulate and control the flow of combustion air, as well as facilitating and promoting the mixing of the combustion air with the vaporized fuel. The combustion chamber is adapted to specifically meter and direct the flow of combustion air so as to maintain the stoichiometric ratio with a given quantity of fuel and optimize the efficiency of the combustion process.

Alcohol-fueled stoves known to prior art may optionally employ a windscreen which may encompass the stove to a greater or lesser extent. However, these windscreens are, by design, description and intent, adapted only to shield the stove from the deleterious effects of crosswinds. Unlike a true combustion chamber, these windscreens are not engineered, adapted nor intended to create a controlled volume wherein the entire combustion process can be contained, regulated and optimized. The prior art does not consider these windscreens to be an integral part of the stove. The prior art consistently refers to these windscreens as ancillary components, being separate from the stove proper—an optional piece to be employed as required to shelter the stove. These windscreens are not intended to be fundamental to the operation of the stove and are not adapted to specifically meter and direct the flow of combustion air so as to optimize the efficiency and control the combustion process.

6.2.2 Fuel Vaporizer

Whereas the combustion chamber is designed to effectively meter and control the flow of combustion air, the efficient operation of the stove also relies on the effective generation and distribution of fuel vapor. The respective volumes of the combustion air and the fuel vapor must be

adapted to alternately achieve a stoichiometric ratio of air to fuel for maximum combustion efficiency, or a rarified ratio of air to fuel for reduced heat output and control over the cooking process.

In order for alcohol fuel to burn, the liquid must be converted into a vapor. Consequently, all alcohol-fueled stoves employ some means by which the liquid alcohol fuel can be vaporized. This typically takes the form of some manner of vessel containing liquid alcohol fuel which is caused to be heated and thereby be converted to vapor. Alternately, some form of wick is used to effect the vaporization of the fuel. The efficiency and rate at which the liquid fuel can be vaporized directly affects both the fuel consumption and the heat output of the stove. Stoves known to prior art do not effect the fuel vaporizing process efficiently and demonstrate excessive thermal losses. These excessive thermal losses serve to both reduce the heat output of the stove, as well as increase the fuel consumption.

The inefficiency of the fuel vaporizing process of stoves known to prior art is caused by several factors. These fuel vaporizing devices typically incorporate an excessive external surface area, both as a consequence of their physical size and dimensions, as well as the deliberate application of fins, ribs or other heat transferring features. This excessive surface area causes the loss of heat to the environment primarily through convection. These fuel vaporizing devices typically incorporate an excessive mass as a consequence of their physical size and dimensions, as well as deliberate application of additional mass. This excessive mass causes the loss of heat to the thermal mass of the fuel vaporizing device. These deliberate applications of excessive surface area and excessive mass result from an improper understanding of the physical processes involved in the fuel vaporization. U.S. Pat. No. 4,164,930 clearly illustrates both of these impediments applied to prior art.

Many stoves known to prior art use some form of wick. The employment of a wick significantly circumscribes the performance of the fuel vaporizer. The limited capacity of the wicking process, combined with both the mass and volume of the wick material, render the use of a wick unsuitable where high heat output, small size and light weight are all required.

In addition to incorporating forms and features that are conducive to the excess loss of heat, stoves known to prior art typically do not employ an effective means for mixing the vaporized fuel thoroughly with the combustion air. If the fuel vapor and combustion air are not thoroughly mixed in the proper area beneath the cooking pot, unburned fuel vapor will escape and both the temperature and efficiency of the combustion process will be reduced. To effectively mix the combustion air and fuel vapor it is necessary to induce turbulence in one or both of the fluids. For maximum combustion efficiency, this turbulence must be created at the confluence of the two fluid flows.

The portable, alcohol-fueled cooking stove of the current invention comprises a fuel vaporizer which is engineered and adapted to minimize the heat lost through the external surface area and mass as far as practicable while maintaining the utility of the stove. The volume of the vessel is specifically adapted to hold approximately two U.S. fluid ounces of fuel. This is the optimal fuel capacity of the fuel vaporizer. This limited volume enables the fuel vaporizer to have minimal size and dimensions. This minimal size and dimensions reduce as far as practicable both the thermal mass and external surface area of the fuel vaporizer. This minimizes the heat which is lost to the thermal mass and the external surface area of the fuel vaporizer, and provides the maxi-

imum amount of energy available to effect the vaporization of the liquid fuel. At the same time two U.S. fluid ounces of fuel provides fuel sufficient enough to cook a typical meal for two people. By holding no more fuel than is required to cook a meal, the thermal mass of the fuel is also minimized. This minimal fuel volume and thermal mass reduces as far as practicable the heat lost to the thermal mass of the fuel. Consequently, the optimized size and dimensions of the fuel vaporizer enable complete utility of the stove while minimizing heat loss.

The fuel vaporizer is also designed to accelerate the flow of the fuel vapor to induce a turbulent flow. By throttling the expanding vapor through an array of small orifices, jets of vapor are ejected from the fuel vaporizer. These jets are positioned and located so as to be injected directly into the convective flow of the combustion air. This maximizes the interaction and mixing between the combustion air and the fuel vapor and creates a very efficient diffusion flame.

6.2.3 Adjustable Heat Output

In addition to achieving a high efficiency with the consequent advantages of high, stable heat output and low fuel consumption, the current invention also incorporates a means for simply and conveniently adjusting the heat output to achieve effective and precise control over the cooking process. This heat adjustment is achieved by metering and controlling the flow of combustion air within the combustion chamber. The flow of combustion air is controlled both in volume and location within the combustion chamber.

The ability to control the cooking process by adjusting the heat output of a stove is a primary attribute that defines the utility, convenience and desirability of the stove. Without the facility for effectively adjusting the heat output, a stove is of limited utility and largely unsuitable for cooking. Petroleum-fueled stoves effect the heat adjustment by directly throttling the fuel flow. This in turn reduces the quantity of combustion air which is entrained by the fuel flow.

Alcohol-fueled stoves known to prior art often incorporate no means at all for adjusting the heat output. Those stoves which do provide a heat adjustment typically employ some type of snuffer in the form of a partial lid, cap or cover that is positioned in such a way to obstruct or interfere with the flow of fuel vapor as it escapes from the fuel vaporizer. This method of adjusting the heat output has several disadvantages. Positioning or adjustment of the snuffer can be a rather awkward undertaking and requires removing the cooking pot or reaching under it. This presents a danger of burning or scalding. Often the cooking pot must be completely removed from the stove and set aside while placing the snuffer. Obstructing the flame in this way also tends to produce a small hot spot on the bottom of the cooking pan and prevents the even distribution of heat for thorough and rapid cooking.

The portable, alcohol-fueled cooking stove of the current invention incorporates a practical, convenient, safe and simple mechanism for regulating the heat output of the stove. This mechanism is quick and easy to operate, and it achieves an effective control of the cooking process. This mechanism is integral with the stove and readily accessible on the outside of the stove. The heat output of the stove can be precisely adjusted without removing the cooking pot or risking burns, scalds or other mishaps.

This adjustment of the heat output is effected by metering and controlling the flow of combustion air both in volume and in location. This metering and controlling of the combustion air flow modulates the combustion process which is in contact with the fuel vaporizer, enabling the temperature

of the fuel vaporizer to be regulated. That is, by reducing the volume and redirecting the location of the combustion air within the combustion chamber the fuel vaporizer can be caused to be cooled. As the fuel vaporizer is cooled, it generates less fuel vapor. This reduced flow of fuel vapor, being conditioned by and adapted to the reduced and redirected flow of combustion air, produces a lower overall heat output for the stove. This means of reducing the flow of fuel vapor, in concert with the reduced and redirected flow of the combustion air, allows an effective, convenient and precise control of the cooking process. This means of controlling the heat output also spreads the heat very evenly throughout the top portion of the combustion chamber and around the bottom and the sides of the cooking pot, enabling foods to be cooked quickly and thoroughly without hot spots or burning.

6.2.4 Portability

In addition to the improvements in the heat output, efficiency, utility and convenience of the current invention, it is also adapted to be small, lightweight and easily portable. The various components are engineered as an integrated unit not only to facilitate the operation of the stove, but also to facilitate its portability and storage.

An important measure of the utility of a backpacking stove is how lightweight, simple and convenient it is to carry. Stoves known to prior art may typically consist of a half-dozen or more parts, with little thought given to their packaging. This can render them bulky and difficult to carry, with the potential for lost or damaged parts and added complexity of setup.

All the components of the current invention nest together when packed, forming a single, compact unit in the shape of a hollow cylinder. This cylinder is proportioned to also contain a fuel bottle, whereby the entire invention is rendered conveniently portable in a sturdy and compact package that weighs only a few ounces and can be comfortably held in the palm of the hand. This package is so efficient that when the fuel bottle is filled with fuel, less than ten percent of the package contains empty space. Being so contained in a single, integrated unit, all of the components are protected from loss or damage, and the stove can be quickly unpacked and set up for use.

7 DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWINGS

FIG. 1 shows the preferred embodiment in a cutaway, elevation view of the assembled stove.

FIG. 2 shows the preferred embodiment in a plan view of the assembled stove.

FIG. 3 shows the preferred embodiment of the fuel vaporizer in a perspective view.

FIG. 4 shows the preferred embodiment in sectional elevation with the stove in use and configured to produce maximum heat output.

FIG. 5 shows the preferred embodiment in sectional elevation with the stove in use and configured to produce minimum heat output.

FIG. 6 shows the preferred embodiment in a cutaway, elevation view of the packed stove.

FIG. 7 shows the preferred embodiment in an exploded, perspective view of the packed stove.

FIG. 8 shows the alternate embodiment of the fuel vaporizer in a plan view.

FIG. 9 shows the alternate embodiment of the fuel vaporizer in a cutaway, sectional elevation view.

FIG. 10 shows the alternate embodiment of the fuel vaporizer in a perspective view.

FIG. 11 shows the alternate embodiment of the fuel vaporizer in sectional elevation when it is in use.

N.B.: The material thicknesses of components shown in section in these drawings are approximately ten-thousandths of one inch. Consequently, the section thicknesses are shown in slightly exaggerated scale to identify the sections.

8 DETAILED DESCRIPTION OF THE INVENTION

8.1 General Configuration and Assembly

FIG. 1 shows a cutaway, elevation view of the preferred embodiment of the assembled stove. FIG. 2 shows a plan view of the preferred embodiment of the assembled stove. FIG. 3 shows a perspective view of the preferred embodiment of the fuel vaporizer.

The preferred embodiment of the current invention includes a combustion chamber 101 which comprises a cylindrical wall 1 approximately six inches in diameter and four inches in height. This cylindrical wall 1 is formed from a thin sheet of aluminum, approximately ten-thousandths of one inch thick, being rolled into a cylinder and attached together at the ends 17. The ends 17 are attached by means of rivets 15 that slide into keyhole slots 16. The combustion chamber 101 is intended to alternately sit upon a base or a supporting surface 99 such that the bottom of the combustion chamber is fully closed and sealed off from the air.

The combustion chamber cylindrical wall 1 is perforated by a plurality of circular ports 13 and 14. These ports 13 and 14 admit combustion air into the combustion chamber 101 and meter and direct the flow of this combustion air. These ports 13 and 14 which perforate the combustion chamber wall 1 are arrayed in two sets.

The first set of ports 13 which perforates the combustion chamber wall 1 is the primary set of air metering ports 13. There are eighteen ports in the primary set of air metering ports 13. Each of the eighteen ports in the primary set 13 is one-half inch in diameter. All of the ports of the primary set 13 are coplanar and are arrayed angularly about the combustion chamber cylindrical wall 1, being located approximately one-half inch above the bottom edge of the combustion chamber 101. Under the influence of the natural convection currents resulting from the combustion process, the primary set of ports 13 is adapted to meter the appropriate volume of air to form a stoichiometric ratio with a specific quantity of fuel vapor. The locations and positions of the primary air metering ports 13 are adapted to provide a controlled and directed flow of combustion air to efficiently mix with and effect the combustion of the fuel vapor.

The second set of ports 14 which perforates the combustion chamber wall 1 is the secondary set of air metering ports 14. There are six ports in the secondary set of air metering ports 14. Each of the six ports in the secondary set 14 is one-quarter inch in diameter. All of the ports of the secondary set are coplanar and arrayed angularly about the combustion chamber cylindrical wall 1, being located approximately two and one-quarter inches above the bottom of the combustion chamber 101. The ports of the secondary set 14 are intended to throttle and restrict the flow of air into the combustion chamber, thereby rarifying the quantity of combustion air within the combustion chamber.

The combustion chamber 101 includes an obturating device 2 whereby alternately either the primary set of air metering ports 13, or the secondary set of air metering ports

14 may be blocked off in part or in whole. By alternately blocking either the primary set 13 or the secondary set 14 of air metering ports, both the volume and the flow patterns of the combustion air within the combustion chamber can be regulated and controlled. By controlling both the volume and flow patterns of the combustion air, the heat output and cooking performance of the stove can be controlled and adjusted.

The obturating device 2 consists of a thin, strong, flexible strip of aluminum approximately ten-thousandths of one inch thick and one and one-quarter inches wide. This aluminum strip is formed into a cylindrical band by attaching the ends of the strip together. The ends of the strip are attached by means of a rivet 15 that slides into a keyhole slot. The cylindrical obturating band 2 thus formed has an inside diameter adapted to fit over the outside diameter of the combustion chamber cylindrical wall 1. The diameter of the obturating band 2 is adapted to fit securely over the combustion chamber cylindrical wall 1, while being loose enough to be readily slid over the combustion chamber cylindrical wall 1.

The combustion chamber 101 includes a means for supporting a cooking pot 7. This pot supporting device comprises two steel rods 3, each rod 3 being approximately five and one-half inches long and one-eighth inch in diameter. Both ends of each rod 3 pass through the combustion chamber cylindrical wall 1 and are supported by the cylindrical wall 1. The rods 3 are arranged parallel and coplanar, being spaced approximately three inches apart. Being so arranged, the rods 3 form a rudimentary grillage upon which a cooking pot 7 may be supported. This grillage is positioned approximately one inch down from the top of the combustion chamber 101.

In conjunction with the combustion chamber 101, the stove comprises a fuel vaporizer 104 such that the combustion chamber 101 and fuel vaporizer 104 are engineered and adapted as an integrated unit—operating in balanced synergy to effect a high heat output and fuel efficiency by maintaining a stoichiometric ratio between the combustion air and fuel.

The fuel vaporizer 104 comprises a vessel fabricated from aluminum in the shape of a shallow, cylindrical cup 4. This shallow, cylindrical, cup-shaped, aluminum vessel 4 is approximately two inches in diameter and is adapted to hold approximately two U.S. fluid ounces of fuel.

This shallow, cylindrical cup 4 includes a double wall 8. By means of this double wall 8 the interior volume of the shallow, cylindrical cup 4 is divided into two chambers 10 and 11. The first chamber is an inner chamber 11. The inner chamber 11 forms a cylinder whose diameter is defined by the inside diameter of the double wall 8. This cylindrical, inner chamber 11 is coaxial and concentric with the shallow, cylindrical cup 4. The second chamber is an outer chamber 10. The outer chamber 10 is annular in form, this annulus being formed by the gap between the outer wall of the combustion chamber 101 and the double wall 8. The outer, annular chamber 10 fully encircles the inner cylindrical chamber 11. Said chambers are separated each from the other by the double wall 8. The chambers have some connection and may enjoy limited intercourse by means of a plurality of small ports 9 at the base of the double wall 8.

The inner, cylindrical chamber 11 has a diameter that is at least eighty percent of the overall diameter of the fuel vaporizer 104. This maximizes, as far as is practicable while maintaining the utility of the stove, the outside surface area of the inner, cylindrical chamber 11. This also minimizes, as far as is practicable while maintaining the utility of the

stove, the thickness of the outer, annular chamber 10. These factors work together to maximize the rate of heat transfer and the efficiency of heat transfer between the inner, cylindrical chamber 11 and the outer, annular chamber 10.

The inner, cylindrical chamber 11 of the fuel vaporizer 104 is uncovered at the top, thereby forming an open chamber. The contents of this open, inner chamber 11 may freely communicate with the air. The outer, annular chamber 10 is covered at the top, thereby forming a closed chamber. This closure is interrupted only by a plurality of small orifices 12. There are approximately twenty two of these orifices 12 arrayed in a circular pattern. These orifices 12 are approximately twenty eight-thousandths of one inch in diameter. The diameter of the circular pattern of these orifices 12 is approximately one-eighth of an inch smaller than the overall diameter of the fuel vaporizer 104.

8.2 Configuration for Maximum Heat Output

FIG. 4 shows a section through the elevation view of the stove as it is set up for cooking use and configured for maximum heat output. A cooking pot 7 is shown placed upon the stove, being supported by the pot supporting rods 3. The cooking pot 7 has an outer diameter which is somewhat smaller than the inner diameter of the combustion chamber 101 such that the cooking pot 7 is able to fit inside the combustion chamber 101. The diameter of the cooking pot 7 is approximately five and five-eighths inches. When the cooking pot 7 is seated upon the pot supporting rods 3 and located within the top of the combustion chamber 101, the top of the combustion chamber 101 is, for most of its area, obstructed and closed off by the cooking pot 7. The only opening of the top of the combustion chamber 101 is an annular gap which exists between the cooking pot 7 and the combustion chamber wall 1. This annular gap has a width of about three-sixteenths of one inch. This annular gap acts as a flue and accelerates the combustion gasses out of the top of the combustion chamber 101. This greatly facilitates and strengthens the natural convection process that draws air into the combustion chamber through the air metering ports 13 and 14. This annular gap also increases the transfer of heat from the combustion gasses to the cooking pot 7. As the combustion gasses are accelerated through the annular gap, they give up heat to the cooking pot 7 as defined by the Bernoulli Principle.

With the stove set up as shown in FIG. 4, the combustion chamber 101 creates a controlled volume wherein the entire combustion process can be enclosed, contained and encompassed. The combustion chamber 101 is placed upon some supporting surface or suitable base 99 such that the bottom of the combustion chamber 101 is fully closed off from intruding airflow. The obturating band 2 of the combustion chamber 101 is positioned to fully expose the primary set of air metering ports 13, thereby occluding the secondary set of air metering ports 14. Alcohol fuel 28 is poured into the inner, cylindrical chamber 11 of the fuel vaporizer 104. By means of the plurality of small ports 9 at the base of the double wall 8, the fuel flows into the outer, annular chamber 10 and seeks a common level within the inner chamber 11 and outer chamber 10. The fuel vaporizer 104 is placed in the center of the combustion chamber 101, being set upon the same supporting surface or suitable base 99 that supports the combustion chamber 101. A cooking pot 7 is placed on the stove, obstructing the top of the combustion chamber 101 except for the annular flue which exists between the cooking pot 7 and the combustion chamber wall 1. The alcohol fuel 28 in the inner, cylindrical chamber 11 of the

fuel vaporizer 104 is ignited and combusts by virtue of its free communication with the air.

The combustion of the fuel 28 in the inner, cylindrical chamber 11 causes the double wall 8 of the fuel vaporizer 104 to be heated. The heating of the double wall 8 causes heat to be conducted into the outer, annular chamber 10 of the fuel vaporizer 104. This heat causes the fuel in the outer, annular chamber 10 to vaporize. Because the geometry of the fuel vaporizer 104 is so adapted to maximize the flow of heat from the inner chamber 11 to the outer chamber 10 and minimize the loss of heat from the exterior surface area and thermal mass, the rate of fuel vaporization in the outer, annular chamber 10 is maximized. The vaporized fuel can only escape from the outer, annular chamber 10 via the plurality of small orifices 12 which interrupt the top closure of the outer, annular chamber 10.

In passing through the plurality of small orifices 12, the fuel vapor is accelerated such that jets of fuel vapor 26 are ejected from the fuel vaporizer 104. By virtue of their location along the top edge of the outer diameter of the fuel vaporizer 104, the jets of fuel vapor 26 are ejected directly into the upwelling convection current of combustion air 30 which has been metered and directed by the combustion chamber 101 through the primary set of air metering ports 13. This forceful convergence of the fuel vapor 26 and combustion air 30 produces effective mixing of the fuel and air and creates a hot, efficient diffusion flame—the design and dimensions of the fuel vaporizer 104 and the combustion chamber 101 being so adapted to operate together in balanced synergy and produce a stoichiometric ratio of fuel and air. The combustion gases are directed against the bottom and around the sides of the cooking pot 7 to maximize heat transfer to the cooking pot 7. In this manner the cooking performance, heat output and fuel efficiency of the stove are maximized.

8.3 Configuration for Minimum Heat Output

Notwithstanding the employment illustrated in FIG. 4, where both the heat output and the efficiency of the stove are maximized, the stove can be easily adjusted to reduce the heat output in order to effectively and conveniently control the cooking process. FIG. 5 shows a section through the elevation view of the stove as it is set up for cooking use and configured for minimum heat output. A cooking pot 7 is shown placed upon the stove, being supported by the pot supporting rods 3. To adjust the heat output of the stove, the obturating band 2 is slid down to cover and block the primary set of air metering ports 13. With the obturating band 2 fully occluding the primary set of air metering ports 13, the secondary set of ports 14 is fully exposed. With the obturating band 2 in this position, the primary set of air metering ports 13 is blocked, such that no air can enter the bottom half of the combustion chamber 101. This creates an anaerobic atmosphere in the bottom half of the combustion chamber 101. The absence of oxygen in this anaerobic zone prevents combustion from occurring in and about the fuel vaporizer 104. Being physically removed from, and beneath, the combustion process 27, the fuel vaporizer 104 is caused to be cooled, thereby reducing the rate of vapor generation and altering the flow pattern of fuel vapor. The quantity of fuel vapor is thereby adapted to, and conditioned by, the reduced volume and redirected flow of combustion air.

The fuel vapor diffuses into the top half of the combustion chamber 101, where it opportunistically mixes with oxygen in this rarified top zone. This forms a diverse and dynamic combustion process 27 which puts out a reduced amount of heat while spreading out within the top of the combustion

chamber **101**. This eliminates hot spots which impair the utility of a stove. In this manner a practical, simple and effective control is achieved over the heat output, the shape and location of the cooking flame, and the overall cooking performance of the stove. This control is also very convenient and easy to use, as the obturating band **2** is readily accessible on the outside of the combustion chamber **101**.

Adjusting the obturating band **2** can be accomplished without interference or removal of the cooking pot **7** simply by tapping it lightly to cause it to slide down and cover the primary set of air metering ports **13**.

Consequently, the control of the stove is such that when the primary set of air metering ports **13** is exposed there is the maximum output of heat from the stove. Simply by sliding the obturating band **2** down to cover the primary set of ports **13**, the secondary set of ports **14** is opened and the heat is quickly controlled and reduced. The primary set of ports **13** might alternately be partially unblocked, allowing small quantities of air into the bottom of the combustion chamber **101**. This provides effective and practical control over a range of heat outputs for improved cooking performance.

8.4 Packaging and Portability

FIG. **6** shows a cutaway, elevation view of the packed stove. FIG. **7** shows a perspective, exploded view of the packed stove. These figures illustrate how the stove is conceived as an integrated unit for packing and carrying. The cylindrical wall of the combustion chamber **101** is formed from a thin, aluminum sheet which is attached together at its ends **17**. The ends of this sheet can be unattached and the thin, aluminum sheet is sufficiently strong and flexible to be wrapped into a coil around the cylindrical fuel vaporizer **104**. Likewise, the obturating band **2** of the combustion chamber **101** is formed from a thin, aluminum strip which is attached together at its ends. The ends of this strip can be unattached and the thin, aluminum band **2** is sufficiently strong and flexible to be wrapped into a coil around the coiled combustion chamber wall **1**. The assemblage thereby constructed is in the form of a hollow cylinder approximately two and one-quarter inches in diameter and four inches in length.

This hollow cylinder is of sufficient diameter and sufficient length to efficiently contain a four fluid ounce fuel bottle **5**, four fluid ounces being sufficient volume of fuel for several days hiking. The cap **6** of the fuel bottle **5** is of such size and geometry as to fit efficiently inside the fuel vaporizer **104**. The pot supporting rods **3** store alongside this cylindrical assemblage. Thus the entire invention is rendered conveniently portable in a sturdy, lightweight and compact package which is completely self contained, comprising the entire stove and the fuel bottle. This package fits in the palm of the hand and weighs only a few ounces.

8.5 Alternate Embodiment of the Fuel Vaporizer

FIG. **8** shows a plan view of an alternate embodiment of the fuel vaporizer **131**. FIG. **9** shows a cutaway elevation view of this alternate embodiment fuel vaporizer **131**, and FIG. **10** shows a perspective view. The alternate embodiment of the fuel vaporizer **131** is adapted to concentrate a large amount of heat on a small, confined chamber **34**. The alternate embodiment fuel vaporizer **131** also surrounds and envelopes this small, confined chamber **34** in such a way that very little heat can escape from it. In this way the alternate embodiment of the fuel vaporizer **131** is capable of achieving a high rate of fuel vaporization with the concatenate effect of a high overall heat output for the stove. The alternate embodiment fuel vaporizer **131** is potentially

capable of even greater heat output than the preferred embodiment **104** of the fuel vaporizer.

The alternate embodiment of the fuel vaporizer **131** comprises a vessel in the shape of a shallow, cylindrical cup **31**. This shallow, cylindrical cup **31** incorporates a double wall **32**. By means of this double wall **32** the interior volume of the shallow, cylindrical cup **31** is divided into two chambers **33** and **34**. The first chamber **34** is an inner chamber. The inner chamber **34** forms a cylinder whose diameter is defined by the inside diameter of the double wall **32**. This cylindrical, inner chamber **34** is coaxial and concentric with the shallow, cylindrical cup **31**. The second chamber **33** is an outer chamber. The outer chamber **33** is annular in form, this annulus being formed by the gap between the cylindrical cup **31** and the double wall **32**. The outer, annular chamber **33** fully encircles the inner, cylindrical chamber **34**. Said chambers **33** and **34** are separated by the double wall **32**. The inner and outer chambers **33** and **34** have some connection and may enjoy limited intercourse by means of a plurality of small ports **35** at the base of the double wall **32**.

The outer, annular chamber **33** is open, being uncovered at the top, thereby forming an open chamber. The contents of this open, outer, annular chamber **33** may freely communicate with the air. The inner, cylindrical chamber **34** is closed, being covered at the top, thereby forming a closed chamber. This closure **41** is interrupted only by a plurality of small apertures **40**. This plurality of small apertures **40** is arrayed radially and angularly about the top closure **41** of the inner, cylindrical chamber **34**.

FIG. **11** shows a section view of the fuel vaporizer **131** in use. When the stove is in use, alcohol fuel **28** is poured into the outer, annular chamber **33** of the fuel vaporizer **131**. By means of the plurality of small ports **35** at the base of the double wall **32**, the fuel flows into the inner, cylindrical chamber **34** and seeks a common level within the inner chamber **34** and the outer chamber **33**. The alcohol fuel **28** in the outer, annular chamber **33** of the fuel vaporizer **131** is ignited and causes combustion **38** by virtue of its free communication with the air.

The combustion **38** of the fuel **28** in the outer, annular chamber **33** causes the double wall **32** of the fuel vaporizer **131** to be heated. Because of this heating action accomplished in the outer, annular chamber **33**, this outer, annular chamber **33** can be referred to as the heating chamber. The heating of the double wall **32** causes heat to be conducted into the inner, cylindrical chamber **34** of the fuel vaporizer **131**. This heat causes the fuel **28** in the inner, cylindrical chamber **34** to vaporize. Because of this vaporizing action accomplished in the inner, cylindrical chamber **34**, this inner, cylindrical chamber **34** can be referred to as the vaporizing chamber. The vaporized fuel can only escape from the inner, cylindrical, vaporizing chamber **34** via the plurality of small apertures **40** which interrupt the top closure **41** of the inner, cylindrical, vaporizing chamber **34**. As the fuel vapor escapes through this plurality of small apertures **40**, it is accelerated into jets **39** thereby entraining and actively mixing with the combustion air.

The alternate embodiment **131** shares various features with the preferred embodiment **104** of the fuel vaporizer, as well as employing important differences. Both embodiments of the fuel vaporizer comprise two, concentric, interconnected chambers separated by a double wall. Both embodiments of the fuel vaporizer employ one open chamber to generate heat by means of open combustion. Both embodiments of the fuel vaporizer employ one closed chamber to generate vaporized fuel and accelerate this vapor into jets.

The essential difference between the preferred and alternate embodiments of the fuel vaporizer is the reversal in the function of the inner and outer chambers. The preferred embodiment fuel vaporizer **104** is adapted to nest efficiently with the other components so that the stove can be stowed in a single, compact unit. This requirement dictates certain aspects of the preferred embodiment fuel vaporizer's **104** form, geometry and dimensions. These dictated aspects of the preferred embodiment fuel vaporizer's **104** form, geometry and dimensions necessitate certain compromises that preclude the total optimization of the fuel vaporizer's performance. The primary compromise involves the heat which is lost from the outer, annular vaporizing chamber **10** through the outside wall of the fuel vaporizer **104**.

The alternate embodiment fuel vaporizer **131** is engineered specifically to maximize the rate of fuel vaporization and the heat output of the stove. The form and geometry of the alternate embodiment are not constrained by packaging requirements or other such limitations. This frees the alternate embodiment fuel vaporizer **131** to be fully optimized for heat output and fuel efficiency. By engulfing the inner, cylindrical vaporizing chamber **34** in combustion **38**, its form and design are adapted to minimize heat loss and produce the highest possible temperatures to vaporize and superheat the fuel **28**. By providing the outer, annular heating chamber **33** with an outer perimeter, more oxygen is available for the combustion process **38** therein. These adaptations enable the alternate embodiment fuel vaporizer **131** to achieve the highest efficiency and heat output.

8.6 Advantages of the Current Invention

8.6.1 Advantages of the Preferred Embodiment

It can be seen that the portable, alcohol-fueled cooking stove of the current invention is novel in that it employs a combustion chamber designed to create a controlled volume wherein the entire combustion process can be enclosed, contained and encompassed, thereby regulating and controlling the flow and mixing of the combustion air with the vaporized fuel. This combustion chamber is engineered and adapted to work in balanced synergy with a fuel vaporizer. In combination, these components operate to maintain a stoichiometric ratio of air and fuel to effect a very efficient and high temperature combustion process. This high heat output and balanced combustion also produce stable performance across a wide range of operating conditions and variations of wind and weather. This employment of a combustion chamber and a fuel vaporizer engineered as a unit operating in balanced synergy to maximize the efficiency, temperature and stability of the combustion and cooking processes is both insightful and unknown to prior art.

It can be further seen that the portable, alcohol-fueled cooking stove of the current invention is novel in that it employs a fuel vaporizer engineered to minimize the loss of energy in the fuel vaporizing process as far as practicable, while maintaining the utility of the stove. This is achieved through minimizing the external surface area and mass of the fuel vaporizer through improvements in the geometry, design and configuration of the fuel vaporizer, while providing optimized fuel capacity for convenient cooking. These improvements in the geometry, design and configuration of the fuel vaporizer produce a significant and measurable increase in the efficiency of the fuel vaporizing process. These improvements in the geometry, design and configuration of the fuel vaporizer, which produce a significant and measurable increase in the efficiency of the fuel vaporizing process, are persistently absent from the prior art.

It can be further seen that the portable, alcohol-fueled cooking stove of the current invention is novel in that it employs a combustion chamber which is provided with a plurality of air metering ports. In conjunction with these air metering ports, the combustion chamber employs an obturating device whereby various of the ports might be alternately blocked or unblocked in order to meter and regulate the volume, flow and location of the combustion air within the combustion chamber. By means of this metering, directing and regulating of the combustion air, the fuel vaporizer can be caused to be cooled, thereby reducing the rate of vapor generation and altering the flow pattern of fuel vapor. In this manner a practical, simple and effective control is achieved over the heat output, the shape and location of the cooking flame, and the overall cooking performance of the stove. This method of control causes the heat to be spread evenly throughout the top portion of the combustion chamber, resulting in a uniform cooking temperature and preventing hot spots that can scorch or burn food. This method of regulating the cooking performance of the stove by means of controlling the volume, flow and location of the combustion air, with the concatenate effect of cooling the fuel vaporizer and reducing the flow of fuel vapor, is deviceful and unique.

It can be further seen that the portable, alcohol-fueled cooking stove of the current invention is novel in that it is conceived as an integrated unit for packing and carrying. The various components of the stove nest together when packed, forming a compact, sturdy unit in the shape of a hollow cylinder, which is also proportioned to contain a fuel bottle. Being thus packed, the entire stove, including fuel, forms a tightly integrated assembly which fits in the palm of the hand and weighs only a few ounces. So efficient is this nesting of components, that when packed with a full fuel bottle the stove has over ninety percent of its volume filled. Such an efficient nesting of the various components of the stove, with the inclusion of the fuel bottle, into a single, compact and sturdy unit is inventive and imaginative.

In addition to the specific features, adaptations and forms that render the current invention both novel and an unobvious improvement over prior art, there is a further, compelling testament to the innovation of the current invention. Alcohol-fueled cooking stoves for backpacking and camping are currently available in well over a dozen forms, designs and concepts—either commercially or through published “Do-It-Yourself” instructions. The current invention exhibits a significantly higher heat output, as measured objectively by the time required to boil a given quantity of water, than any currently published performance specifications for any alcohol-fueled backpacking stove. The stove of the current invention exhibits a significantly lower fuel consumption, as measured objectively by the fuel required to boil a given quantity of water, than any currently published performance specifications for any alcohol-fueled backpacking stove.

In addition to the significant and measurable improvements in heat output and fuel efficiency of the current invention when compared to other alcohol-fueled stoves known to prior art, there is a further, prevailing testament to the ingenuity and improvement of the current invention. The performance of the current invention has been tested in a variety of operating conditions against petroleum-fueled backpacking stoves. The current invention consistently meets or exceeds the overall cooking performance of petroleum-fueled stoves, as measured objectively by the total time required to cook a typical, prepackaged, two serving meal of rice. The current invention consistently meets or

exceeds the fuel efficiency of petroleum-fueled stoves, as measured objectively by the gross weight of fuel required to cook a typical, prepackaged, two serving meal of rice. The current invention also exhibits superior high altitude and cold weather performance than many petroleum-fueled stoves.

8.6.2 Advantages of the Alternate Embodiment

It can be seen that the alternate embodiment of the fuel vaporizer is novel in that the inner, cylindrical, vaporizing chamber is fully encircled by the outer, annular, heating chamber. This configuration causes a greater amount of heat to be concentrated in the inner, cylindrical, vaporizing chamber effecting a high rate of fuel vaporization and high heat output. This adaptation of a fuel vaporizer having two, interconnected chambers whereby the inner, cylindrical, vaporizing chamber is fully encircled by the outer, annular, heating chamber enables levels of convenience, safety and performance which are unknown to prior art.

It must be noted that a superficially similar device is known to prior art. This superficially similar device is known from published specifications to produce a greater heat output when compared to other configurations of alcohol stoves known to prior art. However, the current invention differs fundamentally from the prior art and is a significant improvement over the prior art.

Whereas the current invention employs a single vessel that comprises two, interconnected chambers, the superficially similar device known to prior art employs two, separate vessels. The primary vessel is the vaporizing vessel. This vessel is substantially closed except for a plurality of small apertures and a filling port whereby the vessel is provided with fuel. This filling port is subsequently plugged and the vessel is caused to be heated, whereby the fuel inside the vessel is vaporized and escapes through the plurality of small apertures. The second vessel is employed to heat the primary, vaporizing vessel. The second vessel is in the form of a small tray or saucer. The primary, vaporizing vessel is placed in this small tray or saucer shaped vessel and fuel is poured into this tray or saucer shaped vessel. The combustion of the fuel in this tray or saucer shaped vessel provides the heat for vaporizing the fuel in the primary, vaporizing chamber.

The use of two, separate vessels causes this superficially similar device to be both inconvenient and dangerous. The need to fill two, separate vessels is cumbersome and leads to spillage of fuel. Filling the vaporizing vessel through a small filling port requires the use of a funnel or similar device. In addition, the fuel must be premeasured as the fuel level cannot be seen inside the vessel. The plug for the filling port can be lost, rendering the stove inoperable. Alternately, the vaporizing vessel may have a separate lid which must be removed for filling. This separate lid is inconvenient and cumbersome with the potential to be lost or damaged, rendering the stove inoperable. Finally, as the vaporizing vessel is substantially closed except for the plurality of small apertures, the vaporizing vessel can be overheated and present an explosion hazard.

The current invention, being a single vessel comprising two, interconnected chambers, corrects all of the above shortcomings. Fuel is conveniently poured into the open, outer, annular chamber where its level can be easily seen. Graduations may be provided on the side of the vessel to accurately display the fuel volume. No plugs, funnels or other loose pieces are required. Most important, the inner, vaporizing chamber cannot be overpressurized and cannot explode. As the inner, vaporizing chamber enjoys inter-

course with the open, outer annular chamber by means of a plurality of small apertures at the base of the double wall, if the pressure in the vaporizing chamber becomes too high, it simply displaces liquid fuel and escapes harmlessly. The employment of two interconnected chambers in a single vessel, with the concomitant improvements in the convenience, utility and safety of the current invention, is a significant breakthrough.

8.7 Alternatives Closing

Notwithstanding specific descriptions and details listed herein for the purposes of illustration, it must in no way be construed that these specific descriptions and details in any way limit or circumscribe the scope of the invention. Various specific ramifications of the current invention are anticipated.

It is anticipated that the combustion chamber may be made adjustable in diameter such that cooking pots of various sizes can be properly accommodated.

It is anticipated that various, alternate means may be employed for securing together the ends of the combustion chamber cylindrical wall and likewise the ends of the obturating band.

It is anticipated that the stove may be made larger.

It is anticipated that the stove may employ a windscreen or air jacket to enclose the combustion chamber, thereby providing additional protection from extremes of wind and weather.

It is anticipated that the combustion chamber may incorporate an array of dimples or ribs arranged to facilitate the centering of the cooking pot.

It is anticipated that various forms of shrouding or ducting may be employed to direct the flow of combustion air and facilitate the mixing of the combustion air and fuel vapor.

It is anticipated that the combustion chamber may employ divers means for effecting the metering and control of the combustion air flow.

It is anticipated that the stove may be made of more rigid and heavy material whereby it would be suitable for use when portability is not a primary concern.

It is anticipated that the pot supporting device may take a variety of alternate forms.

It is anticipated that the fuel vaporizer may employ divers means for effecting the vaporizing of the liquid fuel.

It is anticipated that provision might be made for supporting a larger cooking pot, frying pan, skillet, or similar cooking utensil on the stove.

Notwithstanding specific descriptions and details listed herein for the purposes of illustration, and notwithstanding various specific ramifications and embodiments listed above, it must in no way be construed that these specific descriptions, details, embodiments or ramifications in any way limit or circumscribe the scope of the invention.

I claim:

1. A lightweight portable cooking stove comprising the combination of:

a combustion chamber comprising a right circular cylindrical substantially imperforate wall having a predetermined inside diameter, a predetermined outside diameter, a top edge, and a bottom edge;

the bottom edge for engagement with an underlying surface to prevent air from entering the combustion chamber from under the bottom edge of the combustion chamber wall;

a first plurality of relatively large diameter ports extending through the combustion chamber wall at substantially equally spaced intervals along a circle extending

17

parallel to and spaced a predetermined distance above the bottom edge of the combustion chamber wall;

a second plurality of relatively small diameter ports extending through the combustion chamber wall at substantially equally spaced intervals along a circle extending parallel to and a predetermined distance below the top edge of the combustion chamber wall; the first plurality of relatively large diameter ports being separated from the second plurality of relatively small diameter ports by a predetermined longitudinal distance;

obturating device comprising an imperforate right circular cylinder having an inside diameter which is substantially equal to the outside diameter of the combustion chamber wall;

the obturating device being slidably supported on the combustion chamber wall for selectively:

simultaneously closing all of the second plurality of ports extending through the combustion chamber wall while simultaneously opening all of the first plurality of ports extending through the combustion chamber wall; or

simultaneously closing all of the first plurality of ports extending through the combustion chamber wall while simultaneously opening all of the second plurality of ports extending through the combustion chamber wall; or

simultaneously partially opening all of the first plurality of ports and partially opening all of the second plurality of ports extending through the combustion chamber wall;

a fuel vaporizer to be supported on the underlying surface and axially aligned with the combustion chamber wall;

the fuel vaporizer comprising inner and outer right circular cylinders having a common axis and each having an inside surface, an outside surface, and upper end, and a lower end;

a first imperforate first bottom wall extending across the entirety of the lower end of the inner right circular cylinder of the fuel vaporizer to define a first cylindrical shaped fuel receiving chamber;

the inside surface of the second right circular cylinder comprising the fuel vaporizer being spaced apart from the outside surface of the first right circular cylinder thereof to define an annular space therebetween;

a second imperforate bottom wall extending between the lower end of the outside surface of the first right circular cylinder comprising the fuel vaporizer and the inside surface of the second right circular cylinder comprising the fuel vaporizer to define a second annularly shaped fuel receiving chamber extending between the first and second right circular cylinders comprising the fuel vaporizer;

a top wall extending between the upper end of the outside surface of the first right circular cylinder and the upper end of the inside surface of the second right circular cylinder comprising the fuel vaporizer;

the top wall having a plurality of apertures fanned there-through at substantially equally spaced intervals, the top wall extending between the outside surface of the first right circular cylinder and the inside surface of the second right circular cylinder comprising the fuel vaporizer to facilitate the passage of vaporized fuel from the second annularly shaped fuel receiving chamber of the vaporizer through the apertures formed in the top wall;

18

at least one passageway extending between the first cylindrical shaped fuel receiving chamber and the second annularly shaped fuel receiving chamber and located adjacent to the first and second bottom walls of the fuel vaporizer;

a cooking pot comprising a right circular cylindrical side wall having an outside diameter which is smaller than the inside diameter of the combustion chamber;

the cooking pot further comprising a substantially planar bottom wall comprising an extension of and extending substantially perpendicular to the cylindrical side wall of the cooking pot; and

means for supporting the cooking pot on the combustion chamber with the bottom wall of the cooking pot being positioned within the combustion chamber;

whereby air entering the first plurality of ports extending through the combustion chamber wall mixes with vaporized fuel discharged through the apertures in the upper wall of the fuel vaporizer to form a stoichiometric ratio of mixed fuel and air thereby combining maximum heat production with maximum fuel efficiency; the products of combustion engaging the bottom wall and the side walls of the cooking pot to effect maximum heat transfer.

2. A lightweight portable cooking stove comprising the combination of:

a combustion chamber comprising a right circular cylindrical substantially imperforate wall having a predetermined inside diameter, a predetermined outside diameter, a top edge, and a bottom edge;

the bottom edge for engagement with an underlying surface to prevent air from entering the combustion chamber from under the bottom edge of the combustion chamber wall;

a first plurality of relatively large diameter ports extending through the combustion chamber wall at substantially equally spaced intervals along a circle extending parallel to and spaced a predetermined distance above the bottom edge of the combustion chamber wall;

a second plurality of relatively small diameter ports extending through the combustion chamber wall at substantially equally spaced intervals along a circle extending parallel to and a predetermined distance below the top edge of the combustion chamber wall;

the first plurality of relatively large diameter ports being separated from the second plurality of relatively small diameter ports by a predetermined longitudinal distance;

obturating device comprising an imperforate right circular cylinder having an inside diameter which is substantially equal to the outside diameter of the combustion chamber wall;

the obturating device being slidably supported on the combustion chamber wall for selectively:

simultaneously closing all of the second plurality of ports extending through the combustion chamber wall while simultaneously opening all of the first plurality of ports extending through the combustion chamber wall; or

simultaneously closing all of the first plurality of ports extending through the combustion chamber wall while simultaneously opening all of the second plurality of ports extending through the combustion chamber wall; or

simultaneously partially opening all of the first plurality of ports and partially opening all of the second plurality of ports extending through the combustion chamber wall;

19

a fuel vaporizer to be supported on the underlying surface and axially aligned with the combustion chamber wall; the fuel vaporizer comprising inner and outer right circular cylinders having a common axis and each having an inside surface, an outside surface, and upper end, and a lower end; 5

a first imperforate first bottom wall extending across the entirety of the lower end of the inner right circular cylinder of the fuel vaporizer to define a first cylindrically shaped fuel receiving chamber; 10

the inside surface of the second right circular cylinder comprising the fuel vaporizer being spaced apart from the outside surface of the first right circular cylinder thereof to define an annular space therebetween;

a second imperforate bottom wall extending between the lower end of the outside surface of the first right circular cylinder comprising the fuel vaporizer and the inside surface of the second right circular cylinder comprising the fuel vaporizer to define a second annularly shaped fuel receiving chamber extending between the first and second right circular cylinders comprising the fuel vaporizer; 20

a top wall extending across the upper end of the first right circular cylinder comprising the fuel vaporizer;

the top wall having a plurality of apertures formed there-through at substantially equally spaced intervals around a circle concentric with and adjacent to the inside surface of the first right circular cylinder comprising the fuel vaporizer to facilitate the passage of vaporized

20

fuel from the cylindrically shaped fuel receiving chamber of the vaporizer through the apertures formed in the top wall;

at least one passageway extending between the first cylindrically shaped fuel receiving chamber and the second annularly shaped fuel receiving chamber and located adjacent to the first and second bottom walls of the fuel vaporizer;

a cooking pot comprising a right circular cylindrical side wall having an outside diameter which is smaller than the inside diameter of the combustion chamber;

the cooking pot further comprising a substantially planar bottom wall comprising an extension of and extending substantially perpendicular to the cylindrical side wall of the cooking pot; and

means for supporting the cooking pot on the combustion chamber with the bottom wall of the cooking pot being positioned within the combustion chamber;

whereby air entering the first plurality of ports extending through the combustion chamber wall mixes with vaporized fuel discharged through the apertures in the upper wall of the fuel vaporizer to form a stoichiometric ratio of mixed fuel and air thereby combining maximum heat production with maximum fuel efficiency;

the products of combustion engaging the bottom wall and the side walls of the cooking pot to effect maximum heat transfer.

* * * * *